Praxisorientiertes Design, Zuverlässigkeits- und Lebensdauerprognosen Tribologischer Kontakte

an den Antriebsystemen Mediumgeschmierter Pumpentriebwerke,
eines selbstschärfenden Messers und einer Zahnbürste.

Practical Design, Reliability and Service Life Predictions of Tribological Contacts

in Drive Systems of a Medium-Lubricated Pump, a self-sharpening Blade and a Toothbrush.

Dr.-Ing. Michael Gleß ContactEngineering.de, Stuttgart

Design, Reliability and Service Life Predictions

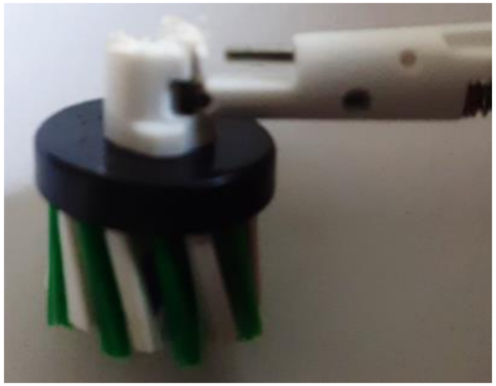
Overview of challenges and in 3 steps to a reliable design

Approach and results in 3 design examples

- Pump drives, Diesel fuel lubricated
  wear-mechanism: **fatigue**
- Control of abrasive **wear**
  => protection and self-sharpening
- Sliding **wear** in gear drives,
  tooth flanks

Pump

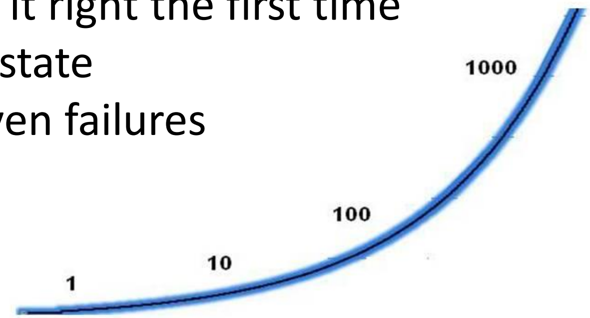
Self-Sharpening agriculture Blade



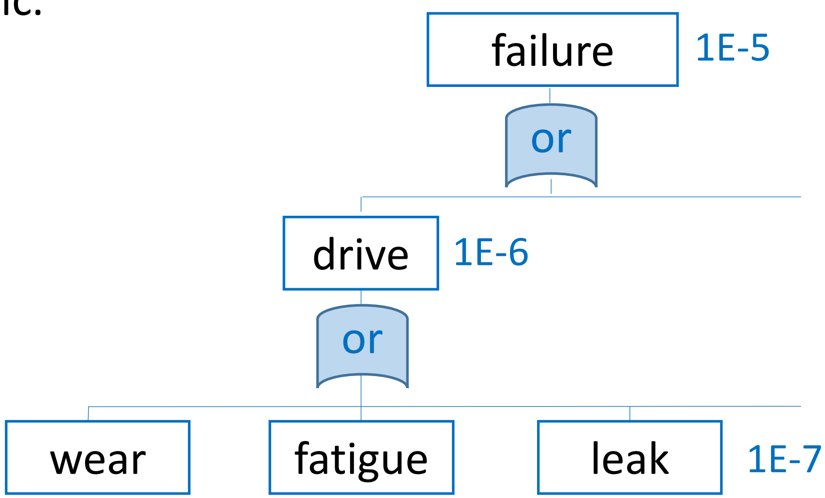
Tooth-brush drive

Challenges in Product Development

- » Design to cost, milestones and timeline targets => do it right the first time
  Choose robust **design** and material selection in early state
  prevent use of non robust designs, late changes or even failures
  (cost rule of 10^{th})
  focus on the correct reproduction of the behavior
  vs. description of all microscopic details
  awareness of tribological complexity
  vs. get lost in details e.g. mixed friction calc.



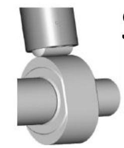
- » Design for Reliability
  service life and **reliability estimation**
  for components and connections
  to reach system reliability target
  vs. missing reliability standards or
  operation outside the
  standardized operating conditions
  e.g. medium lubricated or dry contacts



Design, Reliability and Service Life Prediction Steps

1. Address + verify Requirements and Functions.
   Enables calc./simulation driven development,
   Analysis of **influencing** factors => robust Design

Parameters
dimensions
loads

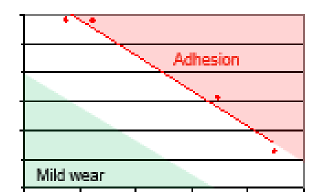


f(x, y)

Service time, reliab.
safety factors
weight, costs

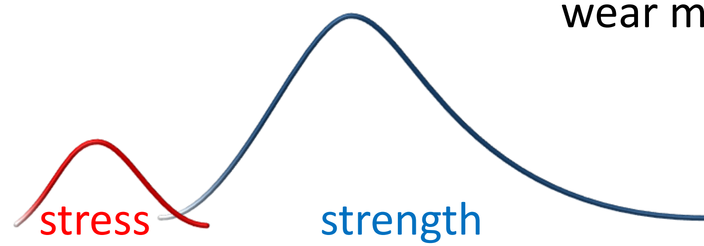
robustness optimize

2. Identify failure/Wear **Mechanism**
   e.g. wear maps and limits in parallel accelerated tests/tribometer
   avoid wear mechanism if possible (robustness, optimization)
   => **focus** on the main service time limiting mechanism in operating



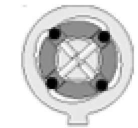
wear map

3. Evaluate resulting stress caused by loads
   vs. strength/resistance (material test)



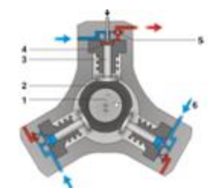
» Estimate **Reliability and Service time.**
  Verify and quality assurance with **Data** from products in production-accompanying testing and field (digital twins)
» Enable a methodical comparison of different **Design** solutions!

1. Drive Design Variants and Tribo-Contact Matrix

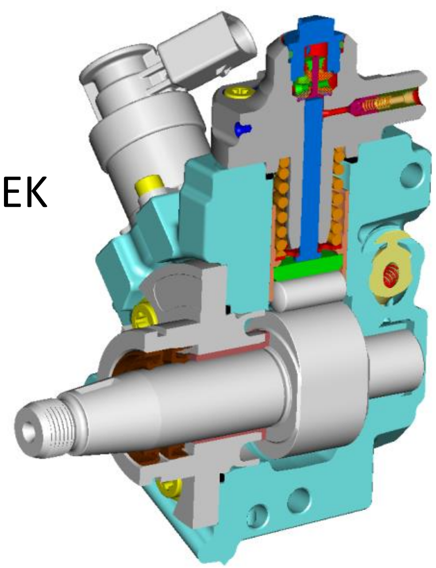


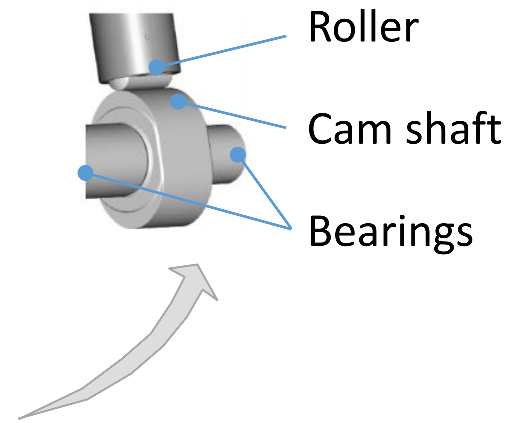
Design Variants comparison

- » complex fuel distributing system in space limited center shaft => wear, cavitation
- » vs. oscillating/reverse sliding contact => high quality surfaces
- » vs. **Cam-Roller Drive** (dynamic motion, less and simpler parts)



Component 1	Component 2	medium diesel fuel
Cam AISI 52100	roller high speed steel/S 6-5-2	
Roller S 6-5	journal bearing	
Cam AISI 52100	journal bearings PEEK	





1. Design of high pressure pumps: Challenges

» Automotive: Development of high pressure pump for 2000 bar+

\+ **pump drive changed from reverse sliding to Cam-Roller contact**, which enables more robust design, low friction, high energy efficiency

\+ complete pump only lubricated with **medium Diesel fuel** therefore more environmental friendly/cleaner combustion without any oil which could enters into fuel

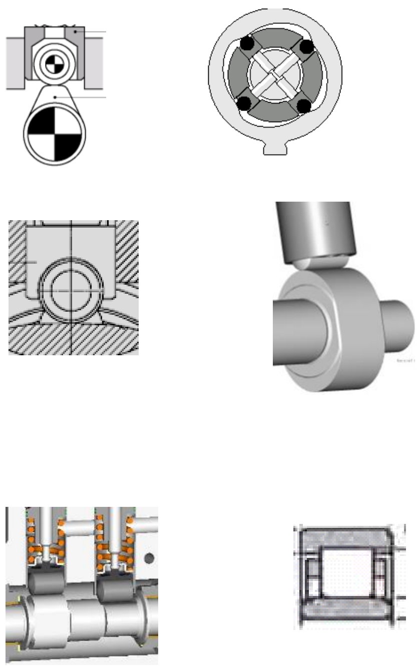
» Required investigation and calculation models of contacts

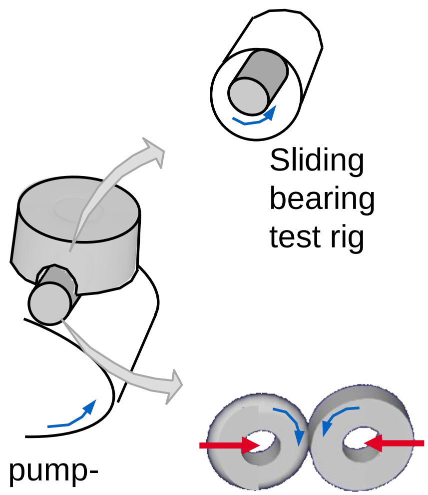
» Lake of understanding for medium lubrication concentrated contact even rolling bearing supplier without knowledge

» **Reliability and service time** of rolling contacts under mixed lubrication condition

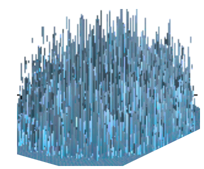
1. Design of high pressure pumps: Synergies

Comparable systems in field	Tribometer-Testing	Model and Simulation

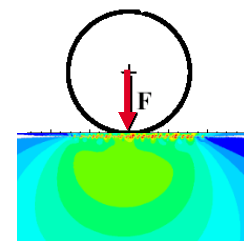




Investigation and analyses of wear mechanisms



Pressure distribution



Stress distribution

Lifetime model

$$\ln\left(\frac{1}{S}\right) \propto N^{e} \int_{V} \frac{(\tau - \tau_{u})^{c}}{z'^{h}} dV$$

Optimization

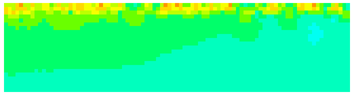
Serice time and reliability prediction und calculation

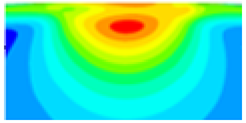
1. Design of high pressure pumps: Influences

Approach of most important stress based influences on **service time**

Analytical | Simulation

→ functions and requirements / ↓ variables and influences	Load-cycles without failure due to fatigue
Design-parameter, design and manufacturing	
Geometry and tolerances	$L \sim p^{-8} \sim \tau^{-8}$
Diameter and tolerance	$L \sim d^{4}$
Length of contact and tolerance	$L \sim l^{4}$
Topographie, roughness, texture, running-in	L = f(macro-, micro-topo.)
Load + operational conditions	
load	$L \sim F_N^{-4}$
friction force	$L \sim \tau^{-8} \approx \tau_{xy}^{-8} \approx (\sigma_x \mu)^{-8}$
Residual stresses (e.g. annealing, treatments, run-in)	$L \sim \tau^{-8}$





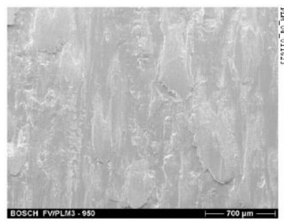
Lifetime model

$$\ln\left(\frac{1}{S}\right) \propto N^{e} \int_{V} \frac{(\tau - \tau_u)^{c}}{z'^{h}} dV$$

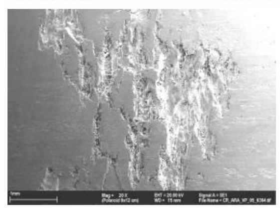
2. Mixed-lubrication failure Mechanism

Working area and limits of a concentrated contact (schematically)

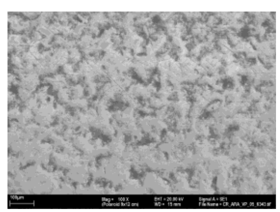
Adhesion

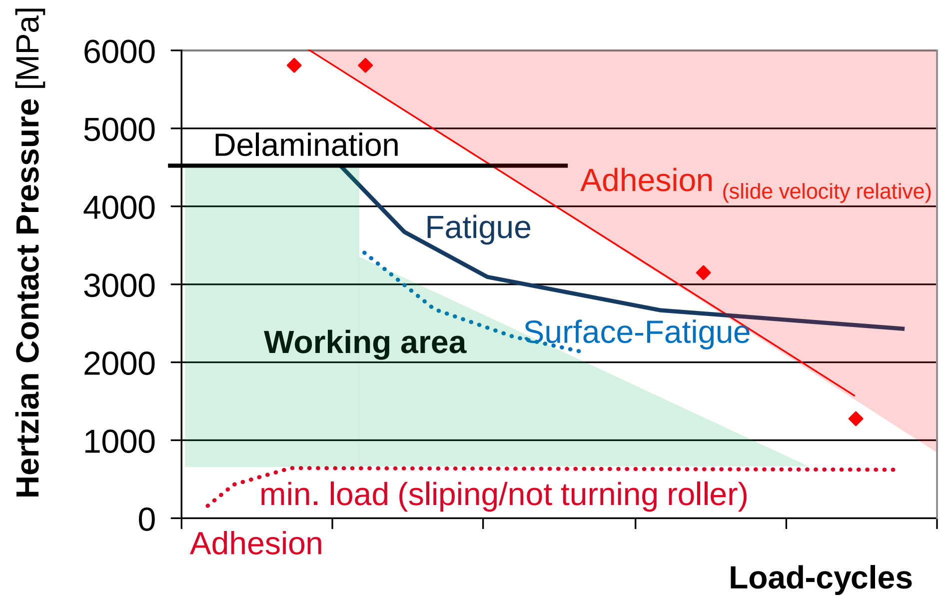


Fatigue
cracks/pitting



Surface-Fatigue
mikro-pitting
mixed lubrication

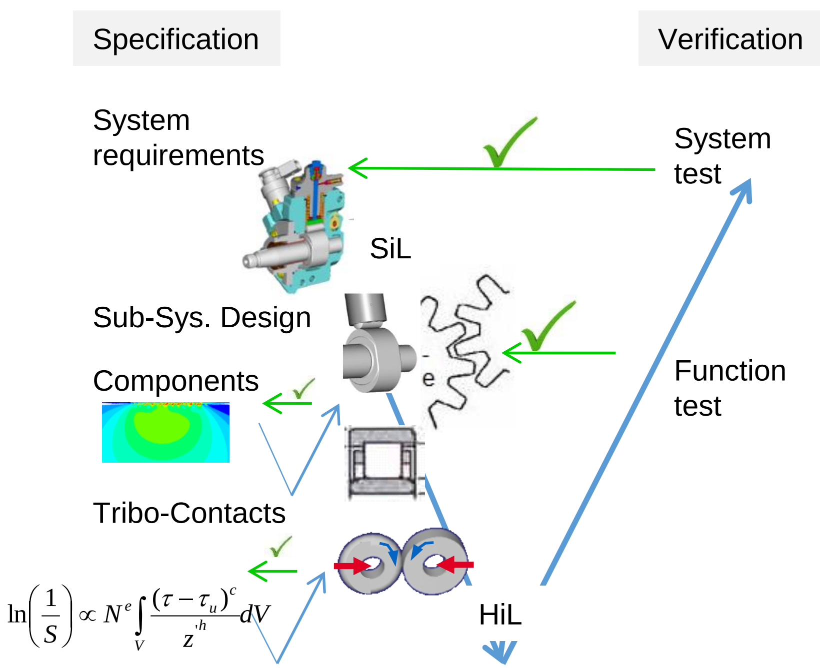


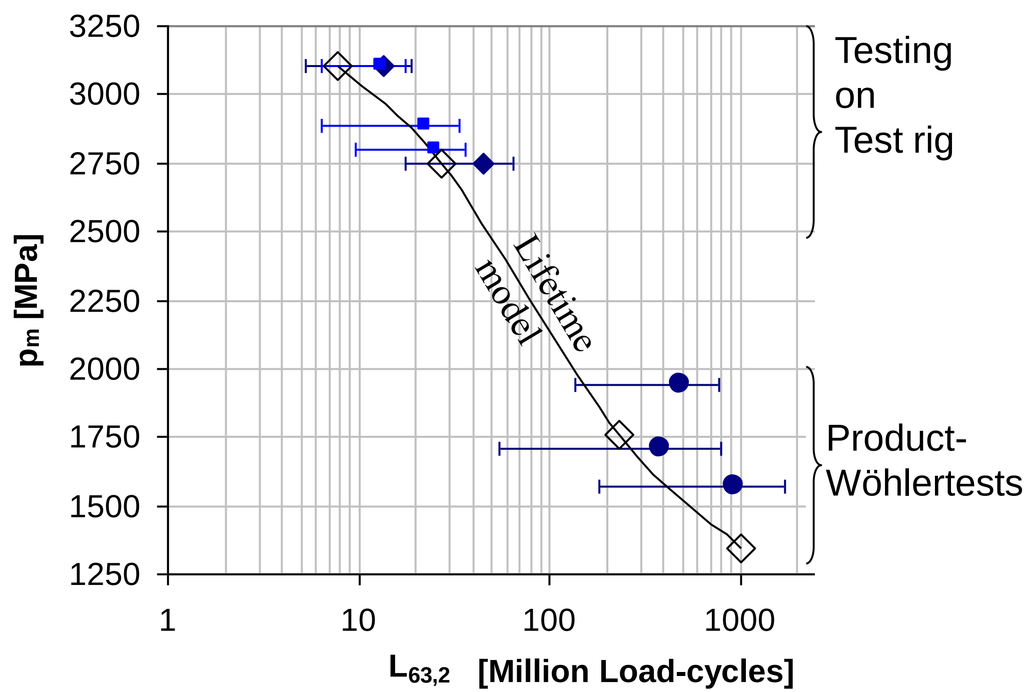


3. Verification and Service life time Model

Concept V-shape model Verification on component level: in the loop MiL, SiL, HiL

1. Evaluation of influences and failures modes
2. Tribometer testing an field Short Tribometer testing to verify strength and material behavior + verification in production accompanying and field data
3. Simulation and life time models => lifetime and reliability





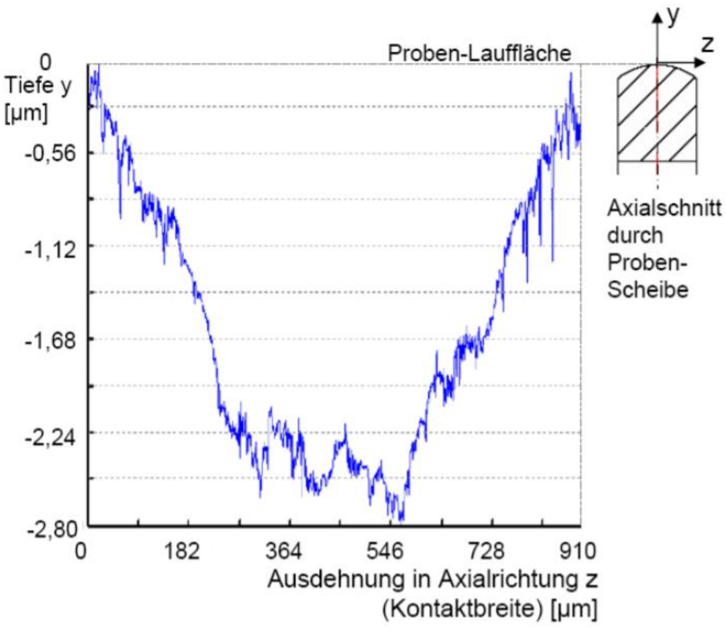
Wear: good or bad?

wear **positive** effects

- Pressure peak reduction, during run process e.g. edges, roughness
- Run in grinding to increase tightness e.g. engine valve seats
- Crack removal / avoid crack propagation e.g. train tracks
- Self sharpening effect e.g. cutting tools, agriculture blades

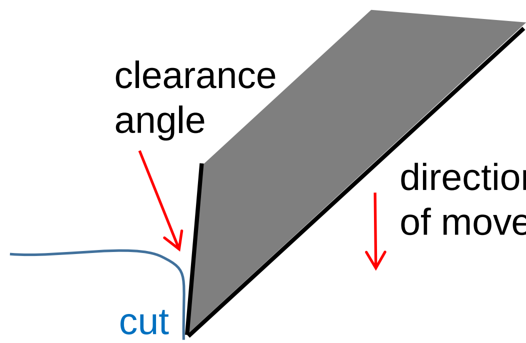
wear **negative** effects

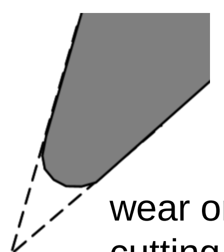
- Loss of material e.g. blades get dull, wear parts
- Loss of isolation e.g. eMobility
- Loss of corrosion protective coating/layers
  e.g. offshore photovoltaic
- Loss of strength, fracture e.g. ICE tire breakage
- Changes in operation e.g. increase of clearance



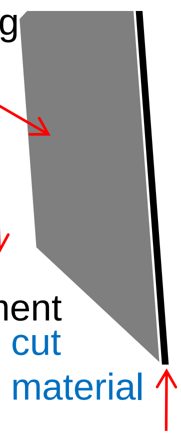
1. Design for abrasive wear Control

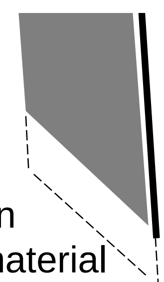
traditional cutting process
with wear on cutting edge

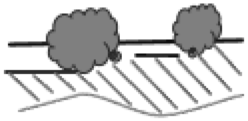




Schematic design based on
bionic principals of a tooth

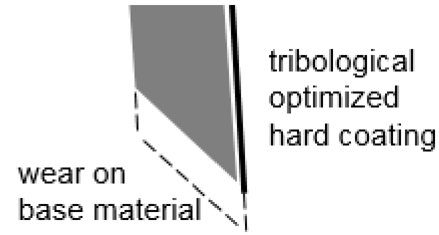


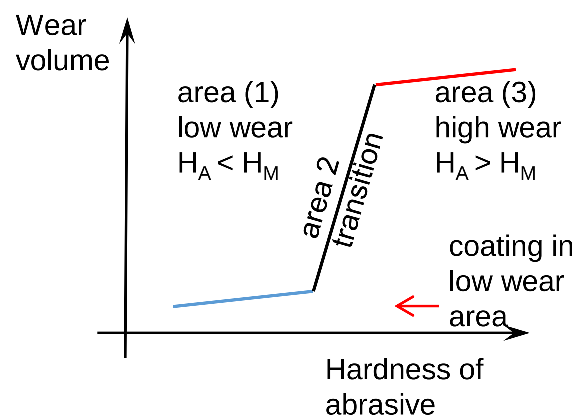




2. Dominating wear mechanism: Abrasion

- Design for self sharpening e.g. agriculture cutting blade
- Dominating wear mechanism: abrasion
- Influences on abrasive wear
  hardness ratio of the coating **material** to the **abrasive** medium / cut material defines the wear area



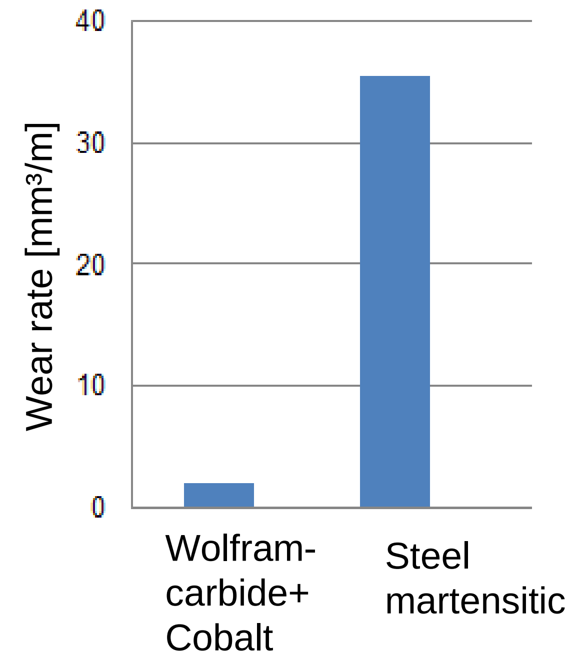


3. Strength and Service life time

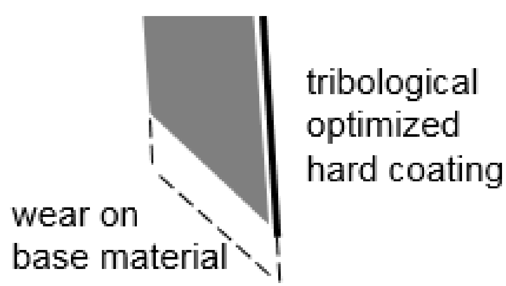
» Strength
  Abrasive Wear Test ASTM G65

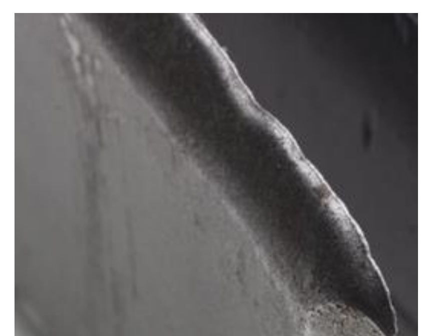
» Coatings
  » Diamond like carbon **DLC**
    Chemical vapor deposition (CVD)
    expensive high end coating

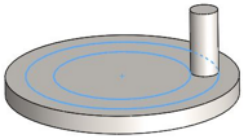
  » **Wolfram-Carbide** on martensitic steel via thermal spray seems to be more economic and offers comparable good results



» Field data
  blade after ~ 350 hours







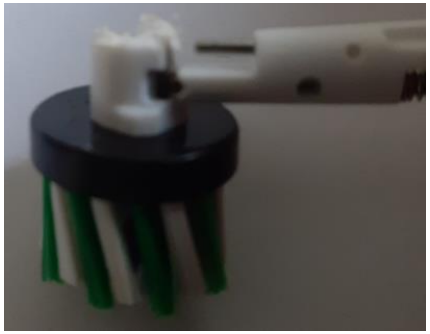
1. Design for service life: Variants and Influences

- Design comparison/selection
  main influencing factors
  - contact pressure and distribution
  - material combination
  - lubricant vs. dry

- Optimization - lower contact pressure, avoid deformation or severe wear

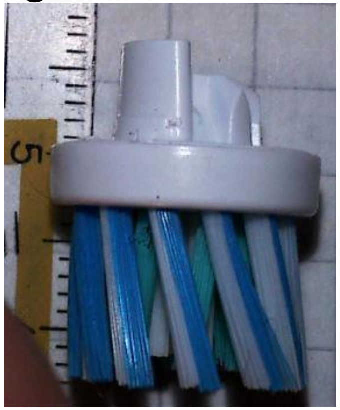
- Mixed lubrication regime,
  mild **wear** is dominating and lifetime limiting

ex-centric

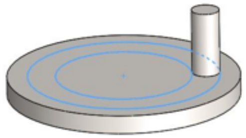


- metal inserts
- many parts
- difficult assembly
+ low friction
+ lifetime

gear drive

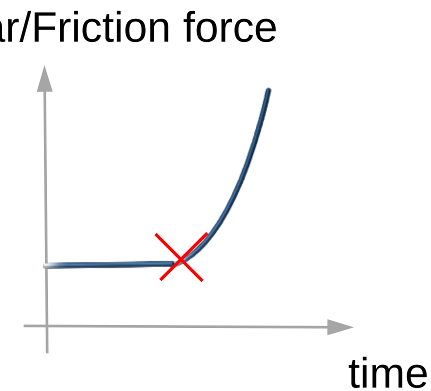


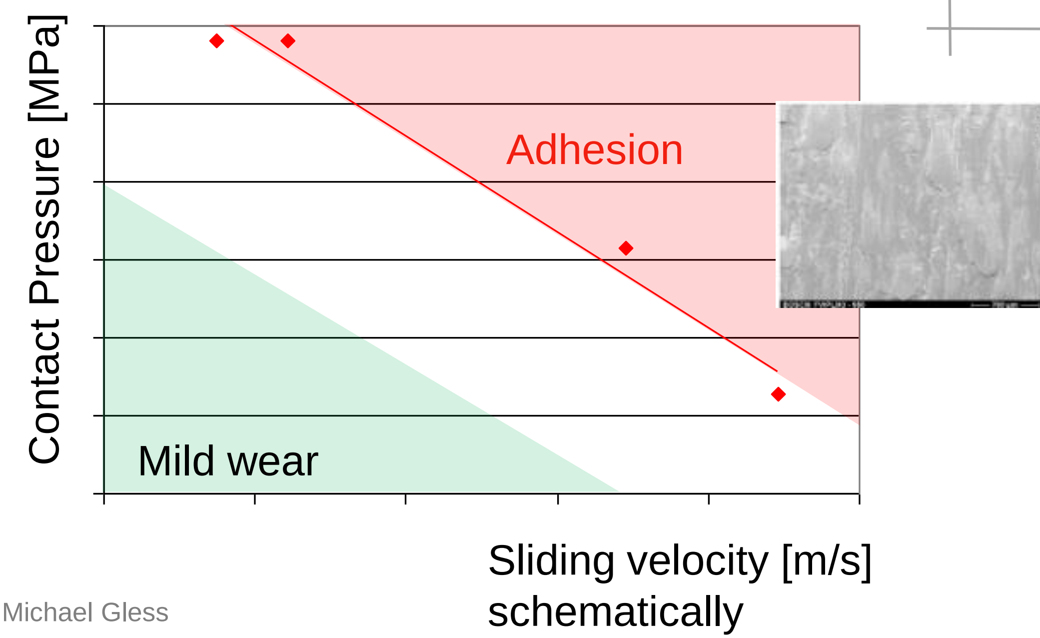
- Plastic material
=> pressure limits
- higher friction
=> higher loads
+ cost

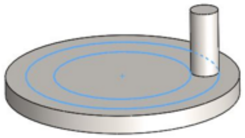


2. Wear Mechanism

» Breakage, adhesion/scuffing, fatigue, …
for design and operation condition
lifetime relevant mechanism is **wear**

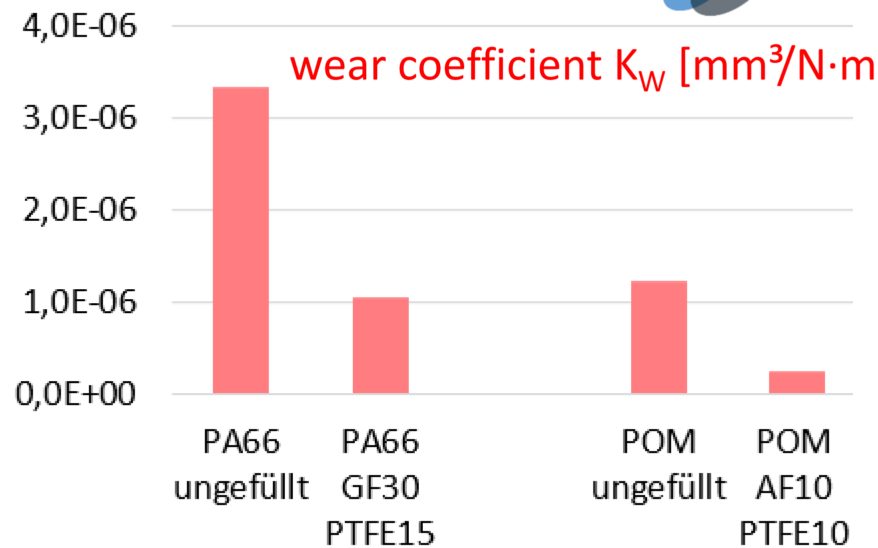


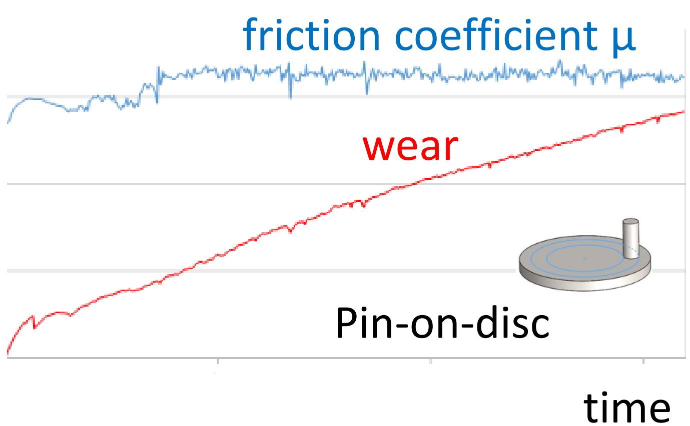


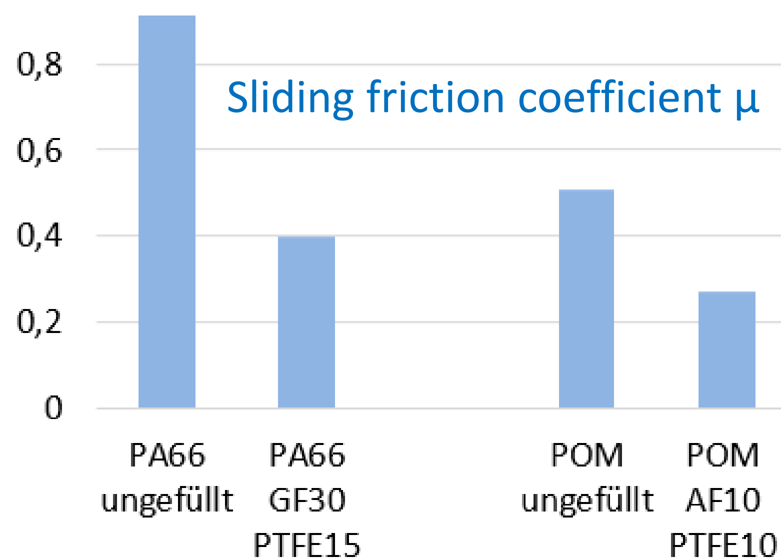


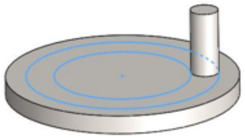
3. Strength and Service time

- Strength/wear resistance
  Derivation max. bearable normal load from experimental determination of wear coefficient K_W as a function of influencing factors (DoE)









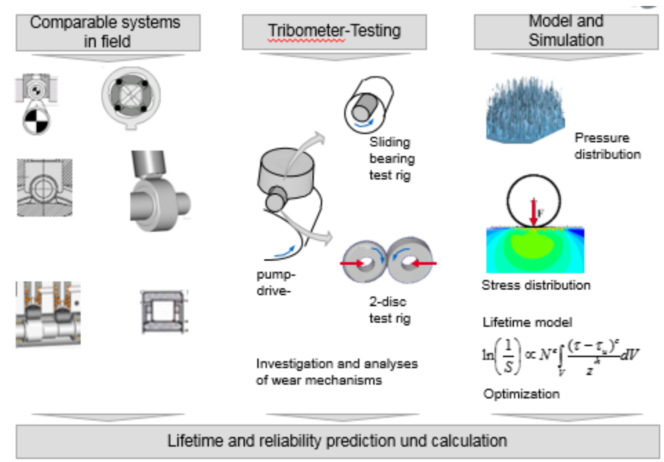
3. Strength and lifetime Wear Prediction

Wear prediction on different Levels

1. Analytical e.g. archard wear, VDI 2736

» within linear wear/steady-state wear regime extrapolation possible

» plastic material suppliers offers wear rates and friction coefficient (pin-on-disc or accord. ASTM)

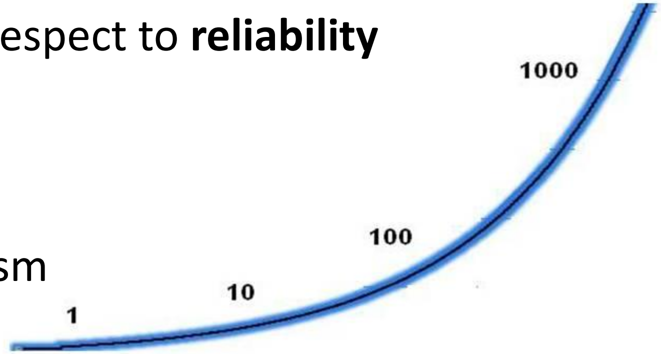


2. Local and iterative e.g. in FEM contact analysis
   load+sliding => wear <=> wear => load/stress

3. adaptive/learning wear prediction method based on continual learning and serial production parallel/accompanying testing

Summary

» An practical driven approach
enables a methodical **comparison** of different design solutions
evaluates design components and influences with respect to **reliability**



» Find robust design and therefore
exclude/avoid wear mechanism and
to focus on/evaluate service time limiting Mechanism

» Tribometer testing supports/verifies Strength and Material behavior.
In addition production accompanying and field **Data** are considered

» Shown approach has been successful applied in several design projects e.g.
to estimate **fatigue** in concentrated contact and
to control **abrasive** wear for self-sharpening and
to estimate sliding **wear** in different contacts.

Thank you for your attention!

ContactEngineering.de
Michael Gleß (Dr.-Ing.)
70437 Stuttgart
Telefon 0711 500 77 433
eMail Contact@**ContactEngineering**.de